

Microstructure Mechanics

Dislocation statics

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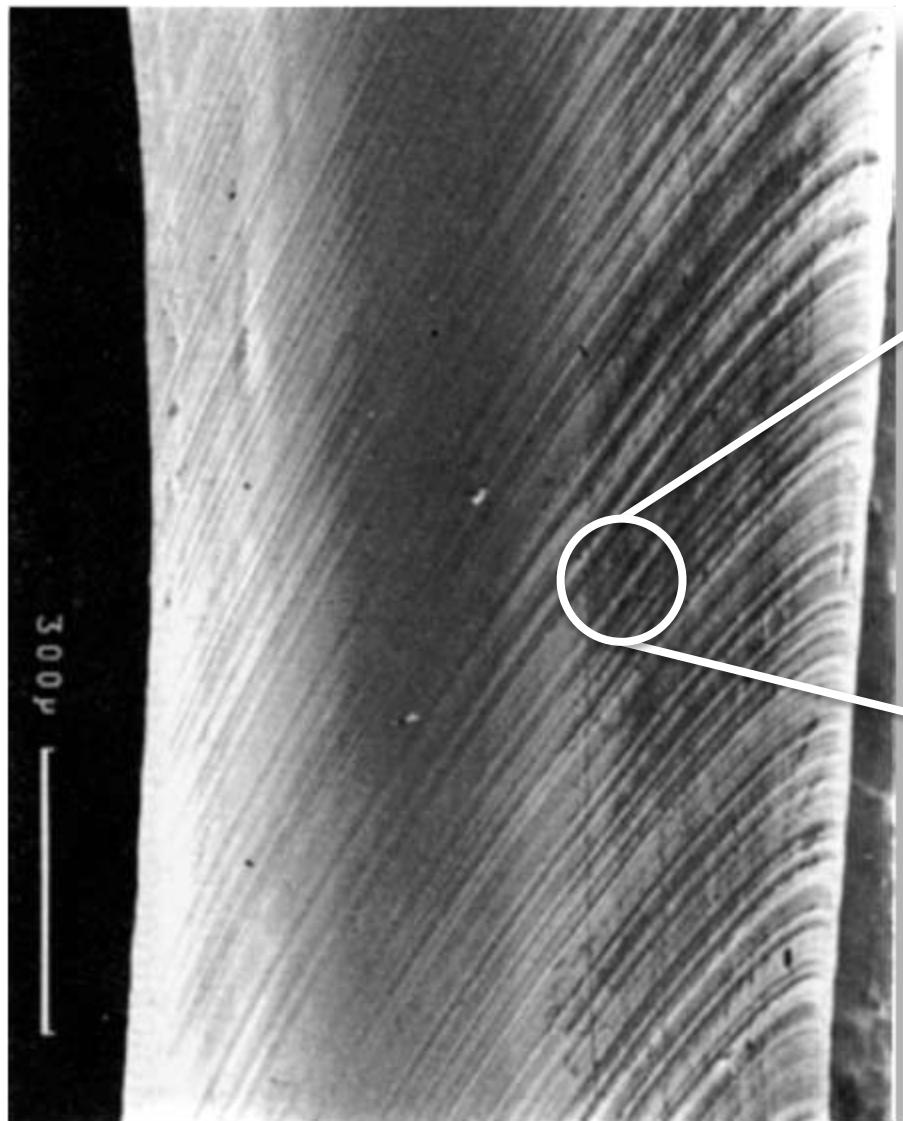


**Max-Planck-Institut
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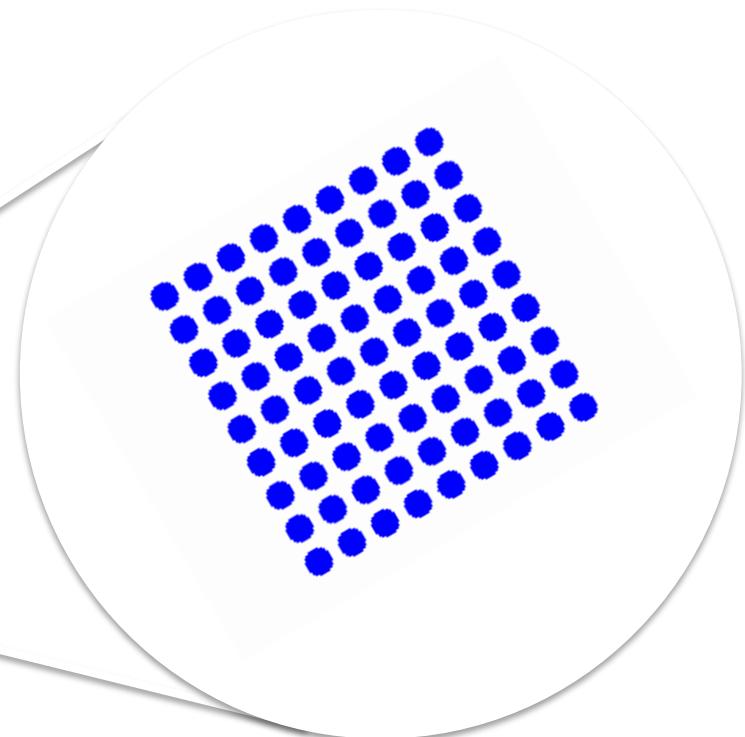
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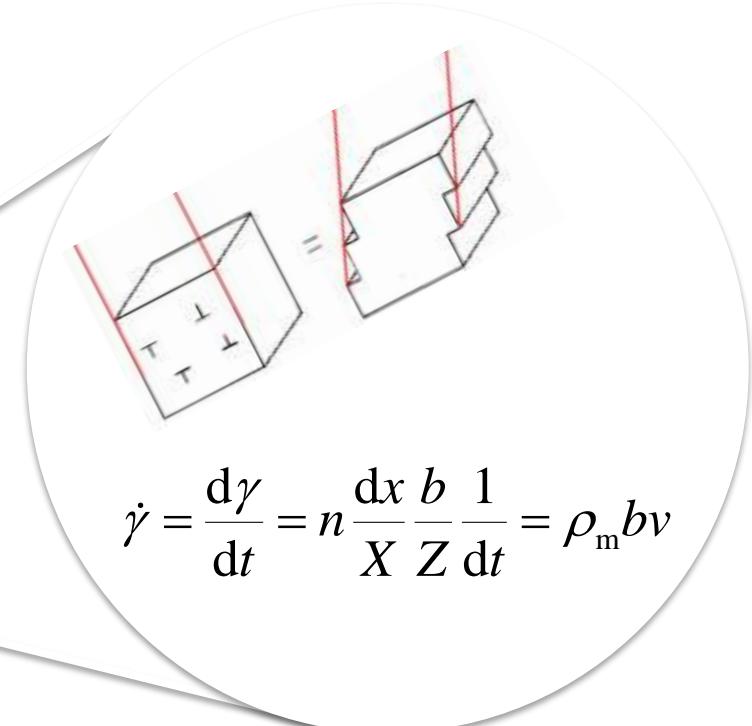
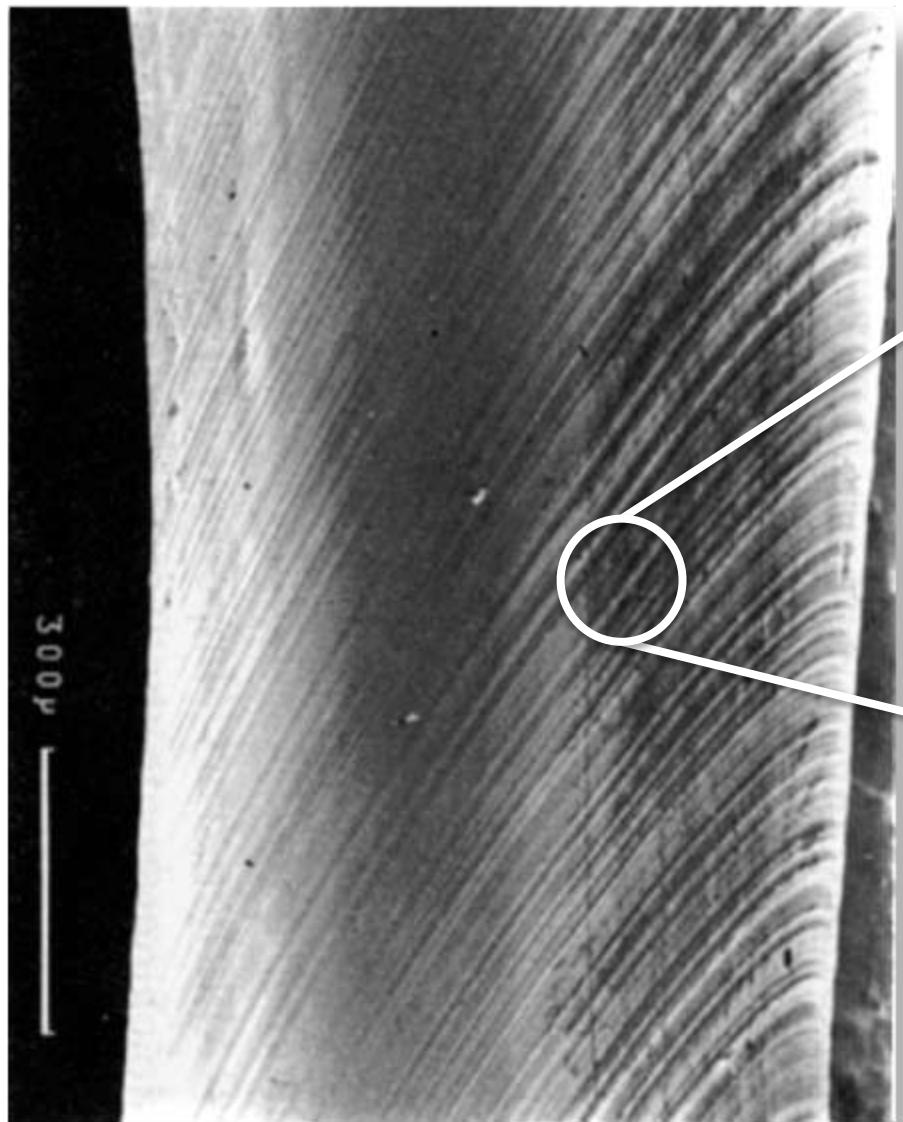


single slip in a single crystal



plastic anisotropy

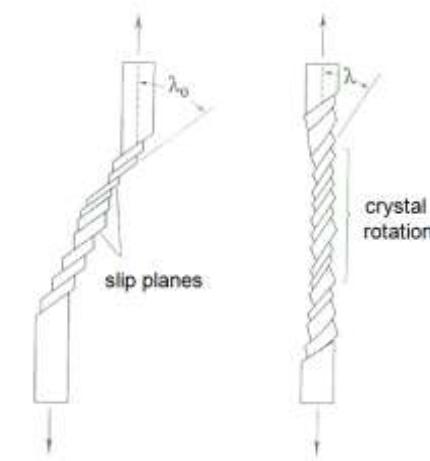
Plastic deformation of a single crystal by dislocation slip



Constraints lead to specific crystal rotations



Non-symmetric dislocation shear leads to rotation



Symmetric-shear can lead to shape change without rotation

Change in local constraints leads to heterogeneity

Kinematics: displacement vector in continuum space



$$\underline{u} = u(x, y, z)$$



$$\underline{u}_{(1)}(x, y, z) = \underline{u}_{(2)}(x, y, z)$$



Kinematics: displacement vector in continuum space



$$\underline{u} = u(x, y, z)$$



$$\underline{u}_{(1)}(x, y, z) \neq \underline{u}_{(2)}(x, y, z)$$





Distorsions come from gradients in the displacement fields

Displacement vector:

$$\mathbf{u} = [u_x, u_y, u_z]$$

Strain tensor:

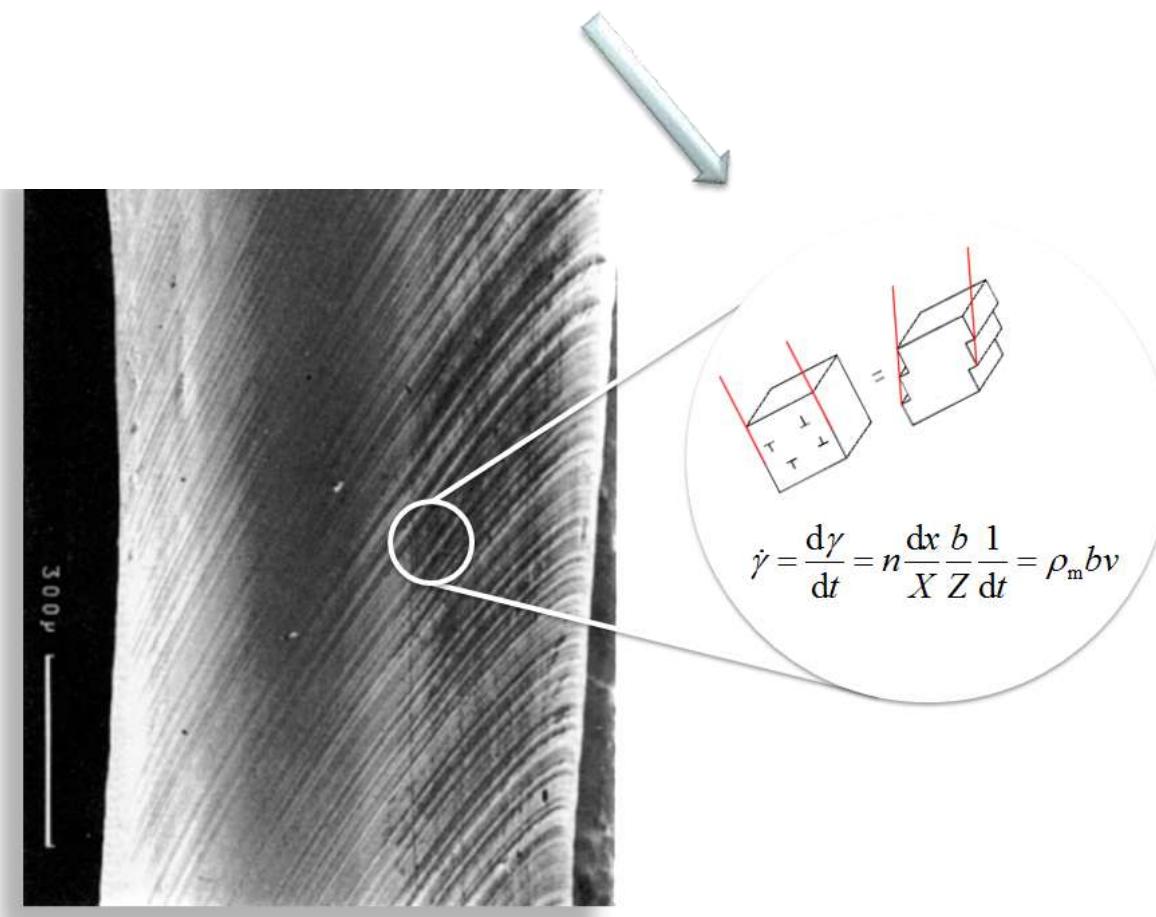
$$\varepsilon_{xx} = \frac{\partial u_x}{\partial x}$$

$$\varepsilon_{xy} = \frac{1}{2} \left(\frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \right)_{etc.}$$

Strain tensor: symmetrical part of displacement gradient tensor

strain rates and displacement gradients in crystals

$$\dot{\varepsilon}_{ij}^K = D_{ij}^K = \frac{1}{2}(\dot{u}_{i,j}^K + \dot{u}_{j,i}^K) = \sum_{s=1}^N m_{ij}^{\text{sym},s} \dot{\gamma}^s \quad \text{mit} \quad m_{ij}^{\text{sym}} = m_{ji}^{\text{sym}} = \frac{1}{2}(n_i b_j + n_j b_i)$$

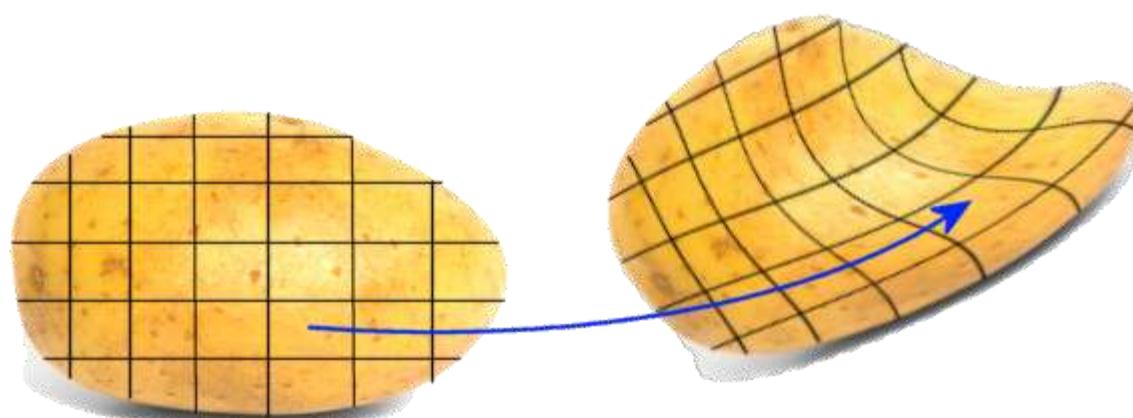


strain rates and displacement gradients in crystals

$$\dot{\varepsilon}_{ij}^K = D_{ij}^K = \frac{1}{2} (\dot{u}_{i,j}^K + \dot{u}_{j,i}^K) = \sum_{s=1}^N m_{ij}^{\text{sym},s} \dot{\gamma}^s \quad \text{mit} \quad m_{ij}^{\text{sym}} = m_{ji}^{\text{sym}} = \frac{1}{2} (n_i b_j + n_j b_i)$$

plastic spin from polar decomposition

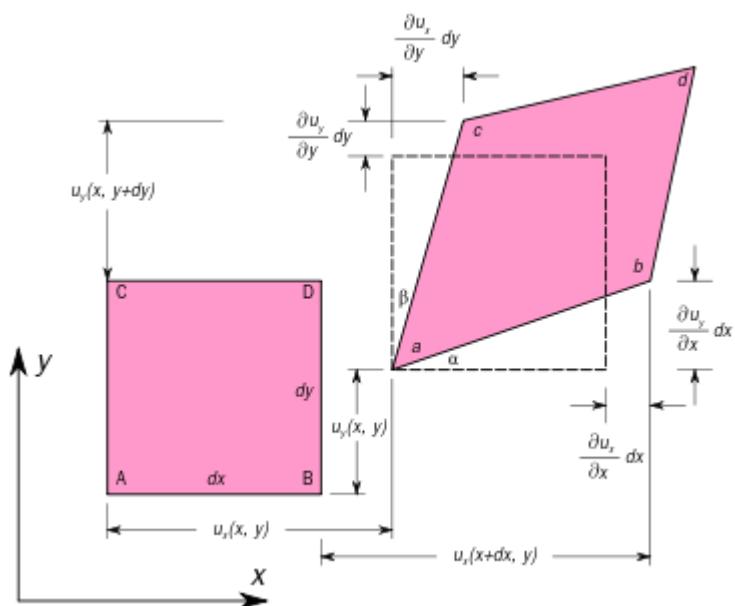
$$\dot{\omega}_{ij}^K = W_{ij}^K = \frac{1}{2} (\dot{u}_{i,j}^K - \dot{u}_{j,i}^K) = \sum_{s=1}^N m_{ij}^{\text{asym},s} \dot{\gamma}^s \quad \text{mit} \quad m_{ij}^{\text{asym}} = -m_{ji}^{\text{asym}} = \frac{1}{2} (n_i b_j - n_j b_i)$$



Displacement gradient

The tensor $\frac{\partial u_i}{\partial x_j}$ is called **displacement gradient tensor** and may be written as

$$\frac{\partial u_i}{\partial x_j} = u_{i,j} = \begin{bmatrix} \frac{\partial u}{\partial x} & \frac{\partial u}{\partial y} & \frac{\partial u}{\partial z} \\ \frac{\partial v}{\partial x} & \frac{\partial v}{\partial y} & \frac{\partial v}{\partial z} \\ \frac{\partial w}{\partial x} & \frac{\partial w}{\partial y} & \frac{\partial w}{\partial z} \end{bmatrix}$$



Displacement gradient tensor

The displacement gradient tensor in general is a non-symmetric tensor and can be decomposed into symmetric and antisymmetric part. Hence the displacement is

$$\begin{aligned} u_i &= \underbrace{u_i^0}_{\text{translation vector}} + \underbrace{\frac{1}{2} (u_{i,j} + u_{j,i}) dx_j}_{\text{strain tensor}} + \underbrace{\frac{1}{2} (u_{i,j} - u_{j,i}) dx_j}_{\text{rotation tensor}} \\ &= u_i^0 + \varepsilon_{ij} dx_j + \omega_{ij} dx_j \end{aligned}$$

Strain tensor

$$\varepsilon_{ij} = \frac{1}{2} (u_{i,j} + u_{j,i})$$

In matrix form

$$\boldsymbol{\varepsilon} = \frac{1}{2} \left(\nabla \mathbf{u} + (\nabla \mathbf{u})^T \right)$$

The above strain tensor is called **Cauchy strain tensor**

$$\begin{aligned}
 \varepsilon_{ij} &= \frac{1}{2} (u_{i,j} + u_{j,i}) \\
 &= \begin{bmatrix} \varepsilon_{11} & \varepsilon_{12} & \varepsilon_{13} \\ \varepsilon_{21} & \varepsilon_{22} & \varepsilon_{23} \\ \varepsilon_{31} & \varepsilon_{32} & \varepsilon_{33} \end{bmatrix} \\
 &= \begin{bmatrix} \frac{\partial u_1}{\partial x_1} & \frac{1}{2} \left(\frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) & \frac{1}{2} \left(\frac{\partial u_1}{\partial x_3} + \frac{\partial u_3}{\partial x_1} \right) \\ \frac{1}{2} \left(\frac{\partial u_2}{\partial x_1} + \frac{\partial u_1}{\partial x_2} \right) & \frac{\partial u_2}{\partial x_2} & \frac{1}{2} \left(\frac{\partial u_2}{\partial x_3} + \frac{\partial u_3}{\partial x_2} \right) \\ \frac{1}{2} \left(\frac{\partial u_3}{\partial x_1} + \frac{\partial u_1}{\partial x_3} \right) & \frac{1}{2} \left(\frac{\partial u_3}{\partial x_2} + \frac{\partial u_2}{\partial x_3} \right) & \frac{\partial u_3}{\partial x_3} \end{bmatrix}
 \end{aligned}$$

Rotation tensor

$$\omega_{ij} = \frac{1}{2} (u_{i,j} - u_{j,i})$$

In matrix form

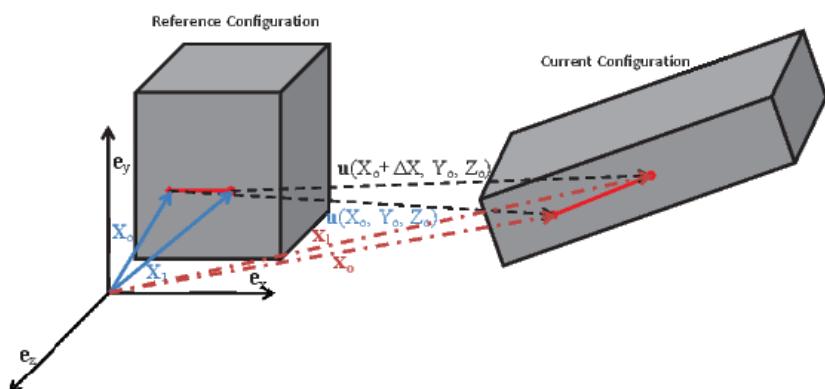
$$\boldsymbol{\omega} = \frac{1}{2} \left(\nabla \mathbf{u} - (\nabla \mathbf{u})^T \right)$$

Matrix expression of the strain tensor

$$\boldsymbol{\varepsilon} = \begin{bmatrix} \varepsilon_{xx} & \varepsilon_{xy} & \varepsilon_{xz} \\ \varepsilon_{xy} & \varepsilon_{yy} & \varepsilon_{yz} \\ \varepsilon_{xz} & \varepsilon_{yz} & \varepsilon_{zz} \end{bmatrix}$$

Matrix expression of the rotation tensor

$$\boldsymbol{\omega} = \begin{bmatrix} 0 & \omega_{xy} & \omega_{xz} \\ -\omega_{xy} & 0 & \omega_{yz} \\ -\omega_{xz} & -\omega_{yz} & 0 \end{bmatrix}$$



Displacement gradient tensor: strain components

Normal strains

$$\varepsilon_{xx} = \frac{\partial u}{\partial x}, \quad \varepsilon_{yy} = \frac{\partial v}{\partial y}, \quad \varepsilon_{zz} = \frac{\partial w}{\partial z}$$

Shear strains

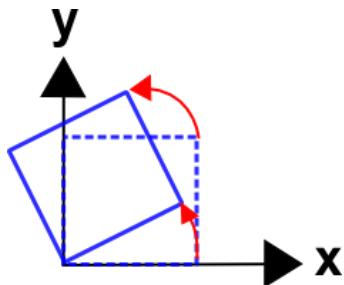
$$\varepsilon_{xy} = \frac{1}{2} \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right)$$

$$\varepsilon_{xz} = \frac{1}{2} \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right)$$

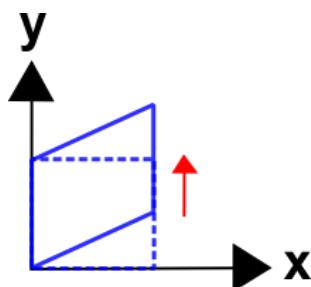
$$\varepsilon_{yz} = \frac{1}{2} \left(\frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \right)$$

Engineering shear strains

$$\gamma_{xy} = 2\varepsilon_{xy}, \quad \gamma_{xz} = 2\varepsilon_{xz}, \quad \gamma_{yz} = 2\varepsilon_{yz}$$

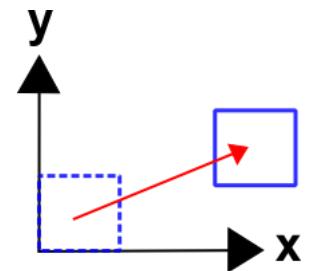


Rigid Body Rotations



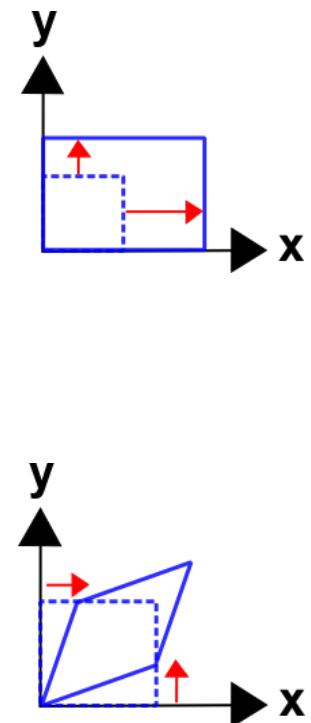
Shear (with Rotation)

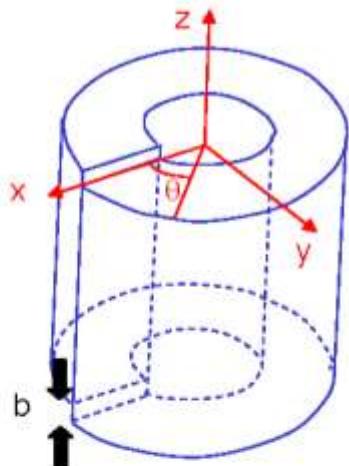
Rigid Body Displacements



Stretching

Pure Shear





"Recipe" :

- take a hollow cylinder, axis along z:
- cut on a plane parallel to the z-axis;
- displace the free surfaces by **b** in the z-direction.

By inspection: $u_x = u_y = 0$

$$u_z = \frac{b\theta}{2\pi}$$

$$= \frac{b}{2\pi} \tan^{-1}\left(\frac{y}{x}\right)$$

$$\begin{aligned}\varepsilon_{xx} &= \varepsilon_{yy} = \varepsilon_{zz} = \varepsilon_{xy} = \varepsilon_{yx} = 0 \\ \varepsilon_{xz} &= \frac{1}{2} \frac{\partial u_z}{\partial x} = \frac{b}{4\pi} \frac{\partial}{\partial x} \tan^{-1}\left(\frac{y}{x}\right) \\ &= -\frac{b}{4\pi} \frac{1}{1 + \left(\frac{y}{x}\right)^2} \frac{y}{x^2} \\ &= -\frac{b}{4\pi} \frac{y}{x^2 + y^2} = -\frac{b \sin \theta}{4\pi r}\end{aligned}$$

$$\begin{aligned}\varepsilon_{yz} &= \frac{1}{2} \frac{\partial u_z}{\partial y} = \frac{b}{4\pi} \frac{\partial}{\partial y} \tan^{-1}\left(\frac{y}{x}\right) \\ &= \frac{b}{4\pi} \frac{1}{1 + \left(\frac{y}{x}\right)^2} \frac{1}{x} \\ &= \frac{b}{4\pi} \frac{x}{x^2 + y^2} = \frac{b \cos \theta}{4\pi r}\end{aligned}$$

Stress field of straight screw dislocation

$$\varepsilon_{xx} = \varepsilon_{yy} = \varepsilon_{zz} = \varepsilon_{xy} = \varepsilon_{yx} = 0$$

$$\varepsilon_{xz} = -\frac{b}{4\pi} \frac{y}{x^2 + y^2} = -\frac{b}{4\pi} \frac{\sin\theta}{r}$$

$$\varepsilon_{yz} = \frac{b}{4\pi} \frac{x}{x^2 + y^2} = \frac{b}{4\pi} \frac{\cos\theta}{r}$$



$$\sigma_{xx} = \sigma_{yy} = \sigma_{zz} = \sigma_{xy} = \sigma_{yx} = 0$$

$$\Delta = 0$$

$$\sigma_{xz} = 2G\varepsilon_{xz} = -\frac{Gb}{2\pi} \frac{y}{x^2 + y^2} = -\frac{Gb}{2\pi} \frac{\sin\theta}{r}$$

$$\sigma_{yz} = 2G\varepsilon_{yz} = \frac{Gb}{2\pi} \frac{x}{x^2 + y^2} = \frac{Gb}{2\pi} \frac{\cos\theta}{r}$$

In Polar coordinates:

(either by direct inspection, or by transforming the strains and stresses from Cartesian co-ordinates)

$$\varepsilon_{\theta z} = \varepsilon_{z\theta} = \frac{b}{4\pi r}$$

$$\sigma_{\theta z} = \sigma_{z\theta} = \frac{Gb}{2\pi r}$$

All other components of the stress tensor are zero.

Note:

- Stress and strain fields are pure shear
- Fields have radial symmetry
- Stresses and strains are proportional to $1/r$:
 - extend to infinity
 - tend to infinite values as $r \rightarrow 0$

Infinite stresses cannot exist in real materials:
the dislocation core radius r_0 is that within which our assumption of linear elastic behaviour breaks down.
Typically $r_0 \approx 1 \text{ nm}$.

Summary: infinite straight screw dislocation

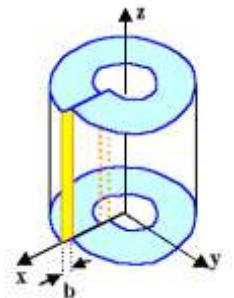
$$\underline{u}(\underline{x}) = \begin{pmatrix} 0 \\ 0 \\ \frac{b}{2\pi} \arctan \frac{y}{x} \end{pmatrix}$$

$$\underline{\underline{\varepsilon}}(\underline{x}) = \begin{pmatrix} 0 & 0 & -\frac{b}{4\pi} \frac{y}{x^2+y^2} \\ 0 & 0 & \frac{b}{4\pi} \frac{x}{x^2+y^2} \\ -\frac{b}{4\pi} \frac{y}{x^2+y^2} & \frac{b}{4\pi} \frac{x}{x^2+y^2} & 0 \end{pmatrix}$$

$$\underline{\underline{\sigma}}(\underline{x}) = \frac{Gb}{2\pi} \begin{pmatrix} 0 & 0 & -\frac{b}{4\pi} \frac{y}{x^2+y^2} \\ 0 & 0 & \frac{b}{4\pi} \frac{x}{x^2+y^2} \\ -\frac{b}{4\pi} \frac{y}{x^2+y^2} & \frac{b}{4\pi} \frac{x}{x^2+y^2} & 0 \end{pmatrix}$$

Summary: infinite straight edge dislocation

$$\begin{aligned}
 u_x &= \frac{b}{2\pi} \left(\arctan \frac{y}{x} + \frac{1}{2(1-\nu)} \frac{xy}{x^2 + y^2} \right) \\
 u_y &= \frac{b}{2\pi} \left(-\frac{1-2\nu}{2(1-\nu)} \log \sqrt{x^2 + y^2} + \frac{1}{2(1-\nu)} \frac{y^2}{x^2 + y^2} \right) \\
 u_z &= 0
 \end{aligned}$$



$$\varepsilon_{xx} = \frac{\partial u_x}{\partial x} = \frac{b y ((3 - 2 \nu) x^2 + (1 - 2 \nu) y^2)}{4 (-1 + \nu) \pi (x^2 + y^2)^2}$$

$$\varepsilon_{yy} = \frac{\partial u_y}{\partial y} = \frac{- (b y ((1 + 2 \nu) x^2 + (-1 + 2 \nu) y^2))}{4 (-1 + \nu) \pi (x^2 + y^2)^2}$$

$$\varepsilon_{xy} = \frac{1}{2} \left(\frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \right) = \frac{b x (-x^2 + y^2)}{4 (-1 + \nu) \pi (x^2 + y^2)^2}$$

Summary: infinite straight edge dislocation

$$\sigma_{xx} = \frac{b G y (3x^2 + y^2)}{2 (-1 + \nu) \pi (x^2 + y^2)^2}$$

$$\sigma_{yy} = \frac{b G y (-x^2 + y^2)}{2 (-1 + \nu) \pi (x^2 + y^2)^2}$$

$$\sigma_{zz} = \frac{b G \nu y}{(-1 + \nu) \pi (x^2 + y^2)}$$

$$\sigma_{xy} = \frac{b G x (-x^2 + y^2)}{2 (-1 + \nu) \pi (x^2 + y^2)^2}$$

Forces and stress field: infinite straight edge dislocation

